



Stochastic dynamics of fluid–structure interaction in turbulent thermal convection

Jinzi Mac Huang¹, Jin-Qiang Zhong², Jun Zhang^{1,3,4}
and Laurent Mertz^{3,†}

¹Applied Math Lab, Courant Institute, New York University, New York, NY 10012, USA

²School of Physics Science and Engineering, Tongji University, Shanghai, 200092, China

³NYU-ECNU Institute of Mathematical Sciences at NYU Shanghai, Shanghai, 200062, China

⁴Department of Physics, New York University, New York, NY 10003, USA

(Received 30 May 2018; revised 15 July 2018; accepted 20 August 2018)

The motion of a free-moving plate atop turbulent thermal convection exhibits diverse dynamics that displays characteristics of both deterministic and chaotic motions. Early experiments performed by Zhong & Zhang (*Phys. Rev. E*, vol. 75 (5), 2007, 055301) found an oscillatory and a trapped state existing for a plate floating on convective fluid in a rectangular tank. They proposed a piecewise smooth physical model (ZZ model) that successfully captures this transition of states. However, their model was deterministic and therefore could not describe the stochastic behaviours. In this study, we combine the ZZ model with a novel approach that models the stochastic aspects through a variational inequality structure. With the powerful mathematical tools for stochastic variational inequalities, the properties of the Markov process and corresponding Kolmogorov equations could be studied both numerically and analytically. Moreover, this framework also allows one to compute the transition probabilities. Our present work captures the stochastic aspects of the two aforementioned boundary–fluid coupling states, predicts the stochastic behaviours and shows excellent qualitative and quantitative agreements with the experimental data.

Key words: convection, flow–structure interactions, turbulent flows

1. Introduction

As an important mechanism of heat transfer in fluids, Rayleigh–Bénard convection (RBC) has been studied extensively in the past (e.g. Ahlers, Grossmann & Lohse 2009). One new aspect of such studies is thermally induced fluid–structure interaction, where buoyancy-driven flow interacts with the solid body and passes momentum

† Email address for correspondence: laurent.mertz@nyu.edu

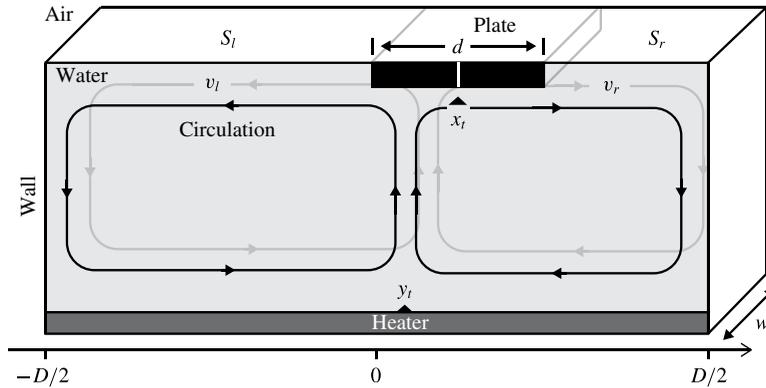


FIGURE 1. Schematic of the system. A floating plate with size d is free to move horizontally on convective fluid in a rectangular tank. The fluid is heated from below and cooled from above, undergoing Rayleigh–Bénard convection and exerting a fluid force on the plate (centre at x_r). Upwelling flow rises from the location y_l and separates into two large-scale circulations that have velocity v_l, v_r to the left and right, respectively.

through fluid forces. Periodic motions and limit cycles are observed in thermally driven systems – for example in Zhang & Libchaber (2000), a plate floating on top of convective fluid undergoes periodic motion between boundaries. Plate tectonics in geodynamics is another example but on a larger scale. Theories suggest that the continental motion is due to the convection in the mantle of Earth (see Turcotte & Schubert 2002), and this theory has been explored both experimentally (Elder 1968; Howard, Malkus & Whitehead 1970; Whitehead 1972) and numerically (Gurnis 1988; Gurnis & Zhong 1991; Lowman & Jarvis 1993, 1995; Whitehead & Behn 2015). Another example is self-propulsion through natural convection (Allshouse, Barad & Peacock 2010; Mercier *et al.* 2014), where an inclined heated surface induces upward buoyancy flow and propels itself through the counterforce generated by the fluid.

Zhong and Zhang have experimentally studied the interaction between convective fluid and a floating plate on the free surface (Zhang & Libchaber 2000; Zhong & Zhang 2005, 2007*a,b*). In their experiment, the authors considered an elongated water tank of length D and width w , as shown in figure 1, where the fluid within is heated uniformly from the bottom and cooled at the open surface. In their experiments, the Rayleigh number $Ra = 1.1 \times 10^9$ and the Prandtl number $Pr = 4.4$, which yield a Reynolds number $Re \sim 3000$ in the bulk. Structures such as thermal plumes are formed and their collective motion forms the large-scale circulation (Krishnamurti & Howard 1981; Ahlers *et al.* 2009). The turbulent aspects of convective fluid, especially with the presence of large-scale circulation, have been modelled as noise in the past (Sreenivasan, Bershadskii & Niemela 2002; Benzi 2005; Brown, Nikolaenko & Ahlers 2005; Zhong, Sterl & Li 2015), and well-developed tools such as the Kolmogorov equations can then be utilized to analyse the stochastic processes involved in RBC. However, when fluid–structure interaction is involved, the existence of solid boundaries complicates the analysis and new mathematical tools are required.

Floating on the open surface, a free-moving plate of length d is subject to the net viscous force from the fluid motion underneath. This mobile plate is free to move along the horizontal direction, with walls on each side of the tank as the limiting boundary. Intriguing dynamics are observed in the experiments, where two states of

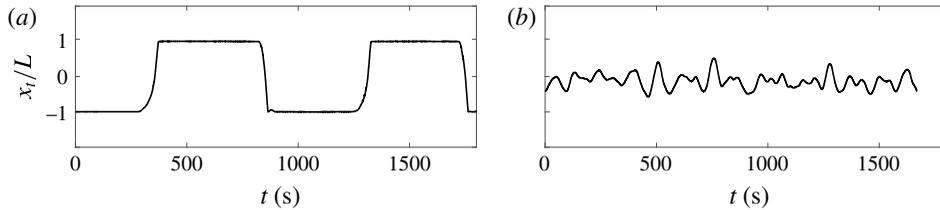


FIGURE 2. Experimental data from Zhong & Zhang (2005) and Zhong & Zhang (2007b). The range of x_r is defined as $L = (D - d)/2$ so $x_r \in [-L, L]$. (a) $d < d_c$: oscillatory state ($d/D = 0.2$), where the floating plate moves towards one sidewall and stays for a random but finite amount of time before moving towards the other sidewall. (b) $d > d_c$: trapped state ($d/D = 0.7$), where the floating plate undergoes stochastic motion at the centre of tank without touching either sidewall.

plate motion exist: when the plate size is smaller than a critical value $d_c = 0.58D$ (Zhong & Zhang 2007b), the plate oscillates between two sides of the tank; as the plate size increases beyond this critical value, the plate becomes trapped in the middle of the tank. Two typical trajectories are presented in figure 2.

Zhong and Zhang also derived a model (ZZ model) in terms of a piecewise smooth dynamical system that incorporates the force balance and the ‘thermal blanket effect’ (Zhang & Libchaber 2000; Zhong & Zhang 2005, 2007a,b). This model explains the existence of two states and provides an explanation of d_c . However, the ZZ model cannot recover any stochastic features observed in the experiment due to its deterministic nature. In reality, the convective fluid is in a turbulent state that generates random forces on the plate, and it is clear that the plate undergoes stochastic motion, as shown in figure 2(b).

Our current work will further advance the ZZ model to a novel variational inequality. Furthermore, in the presence of random fluctuations, the problem becomes a stochastic variational inequality (SVI) whose solution is a continuous (degenerate) reflected diffusion. This type of problem has been studied in the past by Bensoussan *et al.* (2009), Bensoussan & Mertz (2012), Mertz & Feau (2012), Laurière & Mertz (2015), Mertz & Bensoussan (2015), Bensoussan *et al.* (2016), Feau, Laurière & Mertz (2018) and Mertz, Stadler & Wylie (2017), as well as by Bensoussan & Turi (2008) and Bensoussan & Turi (2010) in the context of elasto-plastic oscillators.

Although our model follows along the lines of the ZZ model, the characterization of dynamical states raises completely different problems due to the stochastic nature of our model. One way to treat these problems is to employ the sophisticated (but necessary) mathematical approach based on SVIs. In terms of benefits for the modellers, the SVI approach provides a new tool to access statistics of the system that was not possible before our present work. From the SVI theory, we obtain the Kolmogorov equations that allow us to compute transition probabilities of the stochastic ZZ model, which has an advantage over other stochastic approaches.

As we will show, the good agreement between this newly developed mathematical model and the experiments suggests that the framework of SVI could have further applications in other scientific and engineering fields that involve fluid–structure interactions, where fluid forces are under temporal and/or spatial fluctuations. For example, a limit cycle exists in the geophysical process of continental drift that has been happening constantly on Earth for billions of years. The viscous force on the base of the continents changes the locations of continents in a way similar to the

mechanism depicted in figure 1, and sets up the Wilson cycle of tectonic motion (Turcotte & Schubert 2002).

2. ZZ model and stochastic aspects of the experiments

A physically inspired ODE model is proposed by Zhong and Zhang (ZZ model), and it successfully captures different states of the plate motion. In the experiments, two counter-rotating circulations set up a diverging flow field centred at y_t on the open surface, as shown in figure 1. Depending on its location x_t , the plate is either transported by a single circulation or balanced by the diverging flow field underneath. The circulation speed v_i on the open surface is observed to scale linearly with the open surface area $S_i(x_t, y_t)$ and can be determined as $v_i(x_t, y_t) = v_0 + \theta S_i(x_t, y_t)$, where v_0 and θ are constants and the index $i = (l, r)$ indicates the open surface to the left or the right. In the experiments, a delay $\delta t \approx 10$ s is also observed in the response of flow velocity to the surface area and $v_i(x_t, y_t) = v_0 + \theta S_i(x_{t-\delta t}, y_{t-\delta t})$, which reflects the hysteretic dependence between the flow structure and the plate location. In this study, the delay is ignored due to its relative fast scale compared to other dynamics (typically approximately 5 min), so $S_i(x_{t-\delta t}, y_{t-\delta t}) \approx S_i(x_t, y_t)$ becomes a good approximation in our modelling. As noted in previous works (Zhong & Zhang 2007*b*), the plate dynamics stays unchanged for the model without the delay δt , except for estimating the critical plate size d_c , as we shall discuss later. The net fluid force on the plate can then be associated with the surface flow velocity through integrating the shear stress τ over the plate surface,

$$F(x_t, y_t) = \int_{plate} \tau \, dS \sim \eta w / \lambda \int_{x_t-d/2}^{x_t+d/2} [v_i(x_t, y_t) - \dot{x}_t] \, dx, \quad (2.1)$$

with fluid viscosity η and thermal boundary layer thickness λ . Due to the similar density between the plate and the fluid, the net force $F(x_t, y_t) \approx 0$ leads to the first equation in (2.2). In the experiment, the exposed fluid surface is cooled (i.e. loses heat faster) relative to the covered fluid due to the slowed heat transfer of diffusion through the plate. As a result, heat accumulates under the plate, leading to the ‘thermal blanket effect’ and a local upwelling flow which separates the two large-scale circulations. When the plate is in motion, the upwelling centre y_t follows the plate with a speed that is observed to be proportional to the distance $x_t - y_t$. This attraction sets the motion of y_t and provides the second equation in (2.2) to close the system.

$$\dot{x}_t = -U'_d(x_t; y_t) \quad \text{and} \quad \dot{y}_t = \beta d(x_t - y_t), \quad (2.2a,b)$$

with

$$U_d(x; y) \triangleq \frac{\alpha(2d)}{2} x^2 - \alpha(d)xy, \quad \alpha(\cdot) \triangleq \frac{\theta(\cdot - D) - 2v_0}{d}, \quad \theta, v_0, \beta > 0. \quad (2.3a,b)$$

We have ignored the surface drag term that appeared in Zhong & Zhang (2007*b*) by setting $\gamma = 0$ in their original work. It is noted that the mathematical treatment and results remain the same with or without this surface drag. The model without the surface drag simply suggests that the plate has negligible inertia and is always in equilibrium, such that the net fluid force is 0. Physical parameters θ, v_0, β can be either measured or estimated through experiments, and their values in Zhong & Zhang (2007*b*) are $(\theta, v_0, \beta) = (0.075 \, \text{s}^{-1}, 0.015D \, \text{s}^{-1}, 0.017D^{-1} \, \text{s}^{-1})$.

It is important to emphasize that the potential U_d is unidimensional with respect to the variable x , whereas the variable y appears only as a parameter, with the notation ' representing the derivative with respect to x . To describe the interaction with the walls, let us cite a passage of Zhong & Zhang (2007b):

(BC) The boundary condition is treated as follows: as the floating boundary arrives at the side boundary $x_t = \pm(D-d)/2$, it is set to be at rest at the sidewall $\dot{x}_t = 0$. Meanwhile the underlying flow structure evolves and the horizontal position of the upwelling y_t moves towards the centre of the boundary. The floating boundary remains immobile until the net force from the flows switches direction and starts to drive the floating boundary away from the sidewall.

Physically, the experiment is designed so that the contact between the plate and sidewall is below the fluid–air interface, so the surface tension does not contribute to the boundary interaction and the plate can detach once the sign of U_d is reversed. In the remainder of this study, $L \triangleq (D-d)/2$ so $x_t \in [-L, L]$ and $y_t \in \mathbb{R}$ (y_t is not constrained but remains bounded in $[-D/2, D/2]$ from its own dynamics).

The boundary motion can then be classified into two states according to the shape of the potential in which x evolves, and a direct calculation shows that $U_d''(x; y) = \alpha(2d)$, which determines the convexity of U_d with respect to the direction x . Thus defining $d_c \triangleq D/2 + v_0/\theta$ such that $U_{d_c}''(x; y) = \alpha(2d_c) = 0$, the following properties hold:

- (i) if $d = d_c$, $\alpha(2d_c) = 0$ at criticality, and this is the intermittent state,
- (ii) if $d < d_c$ then $\alpha(2d) < 0$, the potential is concave and leads to an oscillatory state,
- (iii) if $d > d_c$ then $\alpha(2d) > 0$, the potential is convex and leads to a trapped state.

In Zhong & Zhang (2007b), the original ZZ model suggests $d_c = 0.58D$, which accurately captured the critical behaviour. However in our model, the value $d_c = D/2 + v_0/\theta = 0.7D$ is overestimated without the presence of the delay term δt .

In figure 2(b), the stochastic motion of the floating plate is driven by fluctuations in the large-scale circulation, where thermal plumes arrive at the floating plate at random. Due to the stochastic movement of the plume centre y_t , the speed and arrival time of thermal plumes is random and the floating boundary experiences stochastic forcing. This randomness of flow structure inspires us to impose a white noise $\sigma \dot{w}_t$ on the motion of the plumes centre y_t , and (2.2) can be stochastically extended as

$$\dot{x}_t = -U_d'(x_t; y_t) \quad \text{and} \quad \dot{y}_t = \beta d(x_t - y_t) + \sigma \dot{w}_t. \quad (2.4a,b)$$

In a trapped state ($d > d_c$), x_t undergoes stochastic motion without reaching the boundary, and with the structure of (2.4) one can conclude that x_t must have a Gaussian invariant distribution. Indeed this can be seen in figure 3(a), where the histogram of x_t in a trapped state ($d/D = 0.8$) is plotted, and a Gaussian distribution fits well to the experimental data. Knowing the structure of (2.4), the strength of noise σ could be estimated with the standard deviation σ_x of x_t ,

$$\sigma = \sigma_x \sqrt{2\beta d \frac{\alpha(2d)}{\alpha(d)^2} [\alpha(2d) + \beta d] - 2\beta d \frac{\beta d + \alpha(2d)}{\alpha(d)}}. \quad (2.5)$$

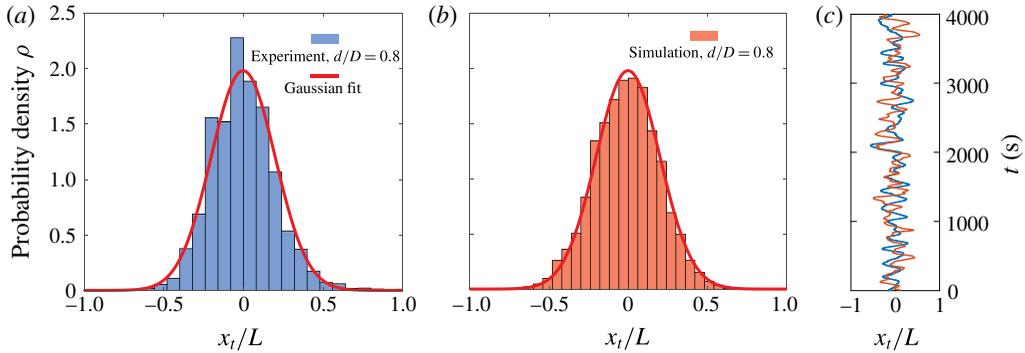


FIGURE 3. PDF and trajectory of the plate centre x_t/L in a trapped state ($d/D = 0.8$). (a) Histogram of x_t from the experiment. A Gaussian distribution is fitted to the data, the mean is near $m = 0$ and standard deviation is $\sigma_x = 0.20$. (b) SDE simulation. The noise in stochastic model is estimated from the experimental data. (c) Trajectories corresponding to (a,b).

From the experimental trajectory of $d = 0.8D$, we found $\sigma = 0.001$. Numerically solving (2.4) with $\sigma = 0.001$ yields a PDF and a trajectory shown in figure 3(b,c), which shows a good agreement with the experiment. This estimation of noise is only valid for the trapped states, where the ZZ model behaves linearly without boundary effects.

3. Mathematical formulation of (stochastic) variational inequality

We can interpret the boundary condition (BC) and (2.2) as follows: the sum of (a) the variation of displacement of the floating boundary, (b) the variation of flow forcing to the plate and (c) the variation of the forces from the wall interaction is 0. That is

$$\text{for all } t_0 < t_1, \quad \underbrace{x_{t_1} - x_{t_0}}_{(a): \text{ displacement}} = - \underbrace{\int_{t_0}^{t_1} U'_d(x_s; y_s) ds}_{(b): \text{ flow}} - \underbrace{(k_{t_1} - k_{t_0})}_{(c): \text{ wall}}. \quad (3.1)$$

To better understand $k(\cdot)$ (k is a time-dependent function), consider the special case of a unilateral constraint $x_t \leq L$ in the sense that the left extremity of the water tank would be located at $x = -\infty$ while the right extremity would remain at L . Then, there is an explicit expression for k_t , that is

$$k_t = \max_{0 \leq s \leq t} \left(x_0 - \int_0^s U'_d(x_s; y_s) ds - L \right)^+, \quad u^+ \triangleq \max(0, u). \quad (3.2a,b)$$

Roughly speaking, dk_t ‘absorbs’ the part of the flow forces that would push x_t beyond the wall. The bilateral constraint case, $|x_t| \leq L$, belongs to the class of the so-called Skorokhod problem – see for instance Lions & Sznitman (1984) and references therein. Adapted to the present problem, it states that for $f_t \triangleq x_0 - \int_0^t U'_d(x_s; y_s) ds \in \mathcal{C}([0, \infty); \mathbb{R})$ with $|x_0| \leq L$, there exists a unique solution (x_t, k_t) of the Skorokhod problem:

$$\begin{cases} x \in \mathcal{C}([0, \infty); [-L, L]), & k \in \mathcal{C}([0, \infty); \mathbb{R}), \\ k \text{ is the sum of two monotonic functions} \\ \text{and } x + k = f. \end{cases} \quad (3.3)$$

Precisely,

$$k_t = \int_0^t \max(0, -U'_d(L, y_s)) \mathbf{1}_{\{x_s=L\}} ds + \int_0^t \min(0, -U'_d(-L, y_s)) \mathbf{1}_{\{x_s=-L\}} ds. \quad (3.4)$$

The last condition means that k_t is increasing, decreasing or constant when $x_t=L$, $x_t=-L$ or $|x_t|<L$, respectively. It is consistent with the description of the wall interaction of the ZZ model since $\mp U'_d(x_t; y_t) \geq 0$ when $x_t = \pm L$.

In this problem, the constraint is given by an interval (convex in \mathbb{R}) and thus it can be recast in terms of a variational inequality (VI) (Bensoussan & Lions 1982) as follows:

$$\text{for all } t \geq 0, \quad |x_t| \leq L, \quad \text{for all } |\xi| \leq L, \quad (\dot{x}_t + U'_d(x_t; y_t))(\xi - x) \geq 0. \quad (3.5a,b)$$

Let us emphasize that, without fluctuations, $\dot{y}_t = \beta d(x_t - y_t)$ indicates that y_t remains in $[-D/2, D/2]$ by its own dynamics. Necessarily, $\pm \dot{y}_t \leq 0$ when $x_t = \pm L$ and $|y_t| \geq L$.

The VI above is an interesting interpretation as it is one way to show that the ZZ model is well-posed in terms of x and y only, which is not obvious. Also, it is useful in the sense that this formulation does not explicitly reveal the terms of the wall interaction but there is a trade-off as we get an inequality.

With the presence of noise, we denote the domain $\mathcal{D} \triangleq (-L, L) \times (-D/2, D/2)$ and thus $\bar{\mathcal{D}} = [-L, L] \times [-D/2, D/2]$. Following the same logic as for equation (2.4), the following SVI can be obtained:

$$\left. \begin{aligned} &\text{for all } t \geq 0, \quad (x_t, y_t) \in \bar{\mathcal{D}}, \quad \text{for all } (\xi, \eta) \in \bar{\mathcal{D}}, \\ &(\dot{x}_t + U'_d(x_t; y_t))(\xi - x_t) + (\dot{y}_t + \beta d(y_t - x_t) + \sigma \dot{w}) (\eta - y_t) \geq 0. \end{aligned} \right\} \quad (SVI)$$

4. Markov properties and Kolmogorov equations

The process (x_t, y_t) is a Markov process on the state space $\bar{\mathcal{D}}$ that is compact. Thus, there exists an invariant probability measure. The semigroup associated to (x_t, y_t) is defined for all $(x, y) \in \bar{\mathcal{D}}$, $\phi \in \mathcal{C}(\bar{\mathcal{D}}; \mathbb{R})$, $t \geq 0$, by $P_t \phi(x, y) \triangleq \mathbb{E}[\phi(x_t, y_t) | (x_0, y_0) = (x, y)]$. Below we use the notation $\mathbb{E}_{(x,y)}[\cdot]$ for $\mathbb{E}[\cdot | (x_0, y_0) = (x, y)]$ and similarly $\mathbb{E}_{(x,y,t)}[\cdot]$ for $\mathbb{E}[\cdot | (x_t, y_t) = (x, y)]$.

4.1. Derivation of the infinitesimal generator

Using Ito's lemma for reflected diffusions, we can derive the infinitesimal generator of (x_t, y_t) . For all functions ϕ that are C^2 w.r.t. to y , C^1 w.r.t. to x , defined on $\bar{\mathcal{D}}$ and $(\partial\phi/\partial y)(x, \pm L) \equiv 0$, we have

$$\phi(x_t, y_t) = \phi(x_0, y_0) + \int_0^t A\phi(x_s, y_s) ds - \int_0^t \frac{\partial\phi}{\partial x}(x_s, y_s) dk_s + \sigma \int_0^t \frac{\partial\phi}{\partial y}(x_s, y_s) dw_s, \quad (4.1)$$

where the differential operator A (degenerate diffusion) is defined by

$$A \triangleq \frac{\sigma^2}{2} \frac{\partial^2}{\partial y^2} + \beta d(x - y) \frac{\partial}{\partial y} - U'_d(x, y) \frac{\partial}{\partial x}. \quad (4.2)$$

From the expression of k_t in (3.4), taking expectation in equation (4.1) gives:

$$P_t \phi(x, y) = \phi(x, y) + \mathbb{E}_{(x,y)} \left[\int_0^t A\phi(x_s, y_s) \mathbf{1}_{\{|x_s|<L\}} ds + \int_0^t B_{\pm} \phi(x_s, y_s) \mathbf{1}_{\{|x_s|=L\}} ds \right], \quad (4.3)$$

where $B_{\pm} \triangleq (\sigma^2/2)(\partial^2/\partial y^2) \pm \beta d(L \mp y)(\partial/\partial y) \pm \min(0, \mp U'_d(\pm L, y))(\partial/\partial x)$. Thus the generator is

$$\lim_{t \rightarrow 0} \frac{1}{t} (P_t \phi(x, y) - \phi(x, y)) = A\phi(x, y)\mathbf{1}_{\{|x| < L\}} + B_{\pm}\phi(x, y)\mathbf{1}_{\{x = \pm L\}}. \tag{4.4}$$

4.2. Kolmogorov equations for the transition probability

Let $0 \leq t < s$ and $(x, y), (\xi, \zeta) \in \bar{D}$. We use the notation $\mu(x, y, t; \xi, \zeta, s)$ for the probability measure of the random walker (plate centre, upwelling plume centre) to be at the position (ξ, ζ) at time s provided that it started from the initial condition (x, y) at time t . An important feature of μ is that it does not have a density with respect to the Lebesgue measure, that makes the problem non-standard. However, it is assumed that μ can be written in terms of three positive and L^1 functions $p(x, y, t; \xi, \zeta, s), p(x, y, t; -L, \zeta, s)$ and $p(x, y, t; L, \zeta, s)$ satisfying

$$\int_{\mathcal{D}} p(x, y, t; \xi, \zeta, s) d\xi d\zeta + \int_{-L}^L p(x, y, t; -L, \zeta, s) d\zeta + \int_{-L}^L p(x, y, t; L, \zeta, s) d\zeta = 1 \tag{4.5}$$

and for any continuous function f on \bar{D} ,

$$\begin{aligned} \mathbb{E}_{x,y,t} f(x_s, y_s) &= \int_{\mathcal{D}} p(x, y, t; \xi, \zeta, s) f(\xi, \zeta) d\xi d\zeta \\ &\quad + \int_{-L}^L p(x, y, t; -L, \zeta, s) f(-L, \zeta) d\zeta \\ &\quad + \int_{-L}^L p(x, y, t; L, \zeta, s) f(L, \zeta) d\zeta. \end{aligned} \tag{4.6}$$

Define the domains $\mathcal{D}_{\pm} = \{\pm L\} \times (-D/2, D/2)$. We use the notation $C_t^1 C_x^1 C_y^2$ for the set of continuous functions on \bar{D} that are differentiable with respect to t, x and twice differentiable with respect to y .

4.2.1. Backward Kolmogorov equation

The equation below is written with respect to the backward variable (x, y, t) . Let $u \in C_t^1 C_x^1 C_y^2$ and assume that it satisfies the following partial differential equation:

$$\left. \begin{aligned} \frac{\partial u}{\partial t} + Au = 0 \quad \text{in } \mathcal{D}, \quad \frac{\partial u}{\partial t} + B_+ u = 0 \quad \text{in } \mathcal{D}_+, \\ \frac{\partial u}{\partial t} + B_- u = 0 \quad \text{in } \mathcal{D}_-, \quad \frac{\partial u}{\partial y} = 0 \quad \text{in } y = \pm \frac{D}{2}, \end{aligned} \right\} \tag{4.7}$$

with the terminal condition $u(x, y, s) = f(x, y)$. Then $u(x, y, t)$ has the following probabilistic interpretation

$$\begin{aligned} u(x, y, t) &= \int_{\mathcal{D}} p(x, y, t; \xi, \zeta, s) f(\xi, \zeta) d\xi d\zeta \\ &\quad + \int_{-L}^L p(x, y, t; -L, \zeta, s) f(-L, \zeta) d\zeta \\ &\quad + \int_{-L}^L p(x, y, t; L, \zeta, s) f(L, \zeta) d\zeta. \end{aligned} \tag{4.8}$$

This can be seen by plugging u in (4.1) and taking expectation $\mathbb{E}_{(x,y,t)}(\cdot)$.

4.2.2. Forward Kolmogorov equation

Although the backward equation can be written explicitly, the complicated boundary behaviours would make the derivation of the forward equation difficult. Usually one can apply integration by parts to transform the backward equation to the forward equation; however, the operators at the boundary B_{\pm} make this procedure impossible. The forward equation on p involves essentially the forward variable (ξ, ζ, s) whereas the backward variable (x, y, t) can be seen as a parameter. The forward Kolmogorov equation for $p(x, y, t; \xi, \zeta, s)$, in an ultraweak sense, is as follows: for any function $\varphi \in C_x^1 C_y^2$

$$\begin{aligned} & \int_{\mathcal{D}} \varphi(\xi, \zeta) p(x, y, t; \xi, \zeta, T) \, d\xi \, d\zeta + \int_{-L}^L \varphi p(\xi, \zeta) p(x, y, t; \pm L, \zeta, T) \, d\zeta \\ &= \varphi(x, y) + \int_t^T \int_{\mathcal{D}} A\varphi(\xi, \zeta) p(x, y, t; \xi, \zeta, s) \, d\xi \, d\zeta \, ds \\ &+ \int_t^T \int_{-L}^L B_{\pm} \varphi(\xi, \zeta) p(x, y, t; \pm L, \zeta, s) \, d\zeta \, ds. \end{aligned} \tag{4.9}$$

One way to derive the forward Kolmogorov equation is to consider an arbitrary smooth function $\varphi(x, y, t)$ and any $T > t$, and write

$$\begin{aligned} \mathbb{E}_{(x,y,t)} \varphi(x_T, y_T, T) &= \varphi(x, y, t) + \mathbb{E}_{(x,y,t)} \left[\int_t^T \left(\frac{\partial \varphi}{\partial t} + A\varphi \right) (x_s, y_s, s) \mathbf{1}_{\{|x_s| < L\}} \, ds \right] \\ &+ \mathbb{E}_{(x,y,t)} \left[\int_t^T \left(\frac{\partial \varphi}{\partial t} + B_{\pm} \varphi \right) (x_s, y_s, s) \mathbf{1}_{\{|x_s| = L\}} \, ds \right]. \end{aligned} \tag{4.10}$$

For the convenience of the reader, we use the notation $p(\xi, \zeta, s) = p(x, y, t; \xi, \zeta, s)$, therefore equation (4.10) can be reformulated as

$$\begin{aligned} & \int_{\mathcal{D}} \varphi p(\xi, \zeta, T) \, d\xi \, d\zeta + \int_{-L}^L \varphi p(\pm L, \zeta, T) \, d\zeta \\ &= \varphi(x, y, t) + \int_t^T \int_{\mathcal{D}} \left(\frac{\partial \varphi}{\partial t} + A\varphi \right) p(\xi, \zeta, s) \, d\xi \, d\zeta \, ds \\ &+ \int_t^T \int_{-L}^L \left(\frac{\partial \varphi}{\partial t} + B_{\pm} \varphi \right) p(\pm L, \zeta, s) \, d\zeta \, ds. \end{aligned} \tag{4.11}$$

Here the typical step to obtain an equation for p with respect to the forward variable (ξ, ζ, s) is to use an integration by parts to put all derivatives w.r.t. to (ξ, ζ, s) on p . For instance,

$$\begin{aligned} \int_t^T \int_{\mathcal{D}} \frac{\partial \varphi}{\partial t} p(\xi, \zeta, s) \, d\xi \, d\zeta \, ds &= - \int_t^T \int_{\mathcal{D}} \varphi \frac{\partial p}{\partial s}(\xi, \zeta, s) \, d\xi \, d\zeta \, ds \\ &+ \int_{\mathcal{D}} \varphi p(\xi, \zeta, T) \, d\xi \, d\zeta \\ &+ \int_{-L}^L \varphi p(\pm L, \zeta, T) \, d\zeta - \varphi(x, y, t). \end{aligned} \tag{4.12}$$

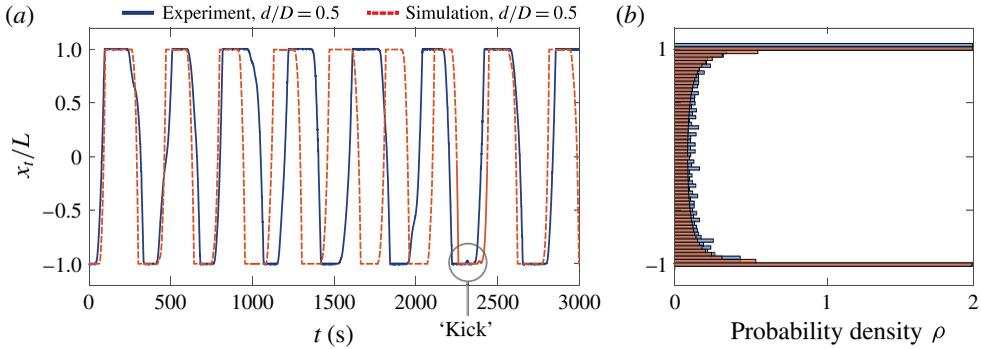


FIGURE 4. Typical trajectory and histogram obtained from experiment and SVI for $d/D = 0.5$. (a) Trajectory extracted from experiment (blue) compared to the SDE simulation (red dashed). The SVI captures the oscillatory behaviour and shows stochastic features such as the ‘kick’ at the boundary. (b) Histogram of the plate centre x_t/L . Simulation and experiment have the same colour scheme as in (a).

A major issue is that this step cannot be done for the integrals on the boundary

$$\left. \begin{aligned} & \int_t^T \int_{-L}^L \max(0, -U'_d(-L, \zeta)) \frac{\partial \varphi}{\partial x} p(-L, \zeta, s) \, d\zeta \, ds \quad \text{and} \\ & \int_t^T \int_{-L}^L \min(0, -U'_d(L, \zeta)) \frac{\partial \varphi}{\partial x} p(L, \zeta, s) \, d\zeta \, ds. \end{aligned} \right\} \quad (4.13)$$

5. Results and discussion

Following the numerical methods in Bernardin (2003), (2.4) can be integrated, and one example trajectory for $d/D = 0.5$ is shown in figure 4(a). Comparing to the experiment, the simulation captures the same dynamics as well as stochastic behaviours. As indicated in figure 4(a), a ‘kick’ can be spotted when the plate is about to leave the wall, and it is a consequence of the noise acting on y_t . As $y_t \rightarrow x_t$, the drift in \dot{x}_t is approaching 0, so the noise could temporarily change the sign of \dot{x}_t and give a brief push to the plate. Moreover, the probability density of x_t can be computed through the statistics of integrating the SDE. Figure 4(b) shows the numerical result compared to the experiment.

One can explore other physical quantities that are susceptible to the noise. When the probability distribution reaches its steady state, the time average of physical measurement should agree with its steady state expectation value obtained in the backward PDE, and here we can demonstrate such agreement with several examples. In figure 5, PDE quantities are obtained by finding $\lim_{s \rightarrow \infty} u(x, y, 0)$ where u solves (4.7) (the dependency w.r.t. s comes from the terminal condition $u(x, y, s) = f(x, y)$). Numerically (Mertz *et al.* 2017), we solve the stationary version of (4.7),

$$\left. \begin{aligned} \lambda u_\lambda - A u_\lambda &= f \quad \text{in } \mathcal{D}, & \lambda u_\lambda - B_+ u_\lambda &= f \quad \text{in } \mathcal{D}_+, \\ \lambda u_\lambda - B_- u_\lambda &= f \quad \text{in } \mathcal{D}_-, & \frac{\partial u_\lambda}{\partial y} &= 0 \quad \text{in } y = \pm \frac{D}{2}. \end{aligned} \right\} \quad (5.1)$$

Here $\lambda > 0$ is a small parameter replacing the time. It can be shown that

$$\lim_{\lambda \rightarrow 0} \lambda u_\lambda(x, y) = \lim_{s \rightarrow \infty} u(x, y, 0). \quad (5.2)$$

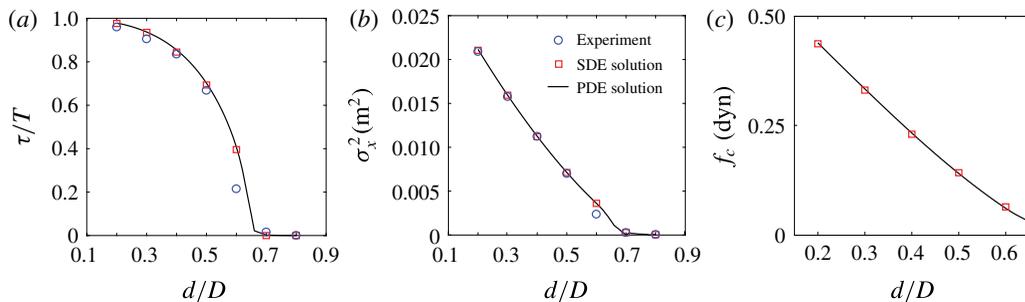


FIGURE 5. Physical quantities obtained from the SVI model compared to the measurements; all quantities are measured and computed in the steady state of statistics. (a) The portion of time that the plate stays at the boundary. Monte Carlo simulation (SDE solution) and the solution of the Kolmogorov equation (PDE solution) agree with experimental measurements. (b) Variance of the plate position can be accurately predicted by the SVI model. (c) Mean contact force computed through the SVI model, which is not yet available from experiments.

Figure 5(a) shows the portion of time that the plate stays at the boundary, which can be obtained through setting $f(x, y) = \mathbf{1}_{\{|x|=L\}}$. There is a clear trend that small plates tend to stay at the boundary longer. There appears to be a deviation at $d/D = 0.6$, which could be due to the fact that the plate size is close to the critical value d_c . Another quantity we can measure is the steady state variance of the plate centre x_t , with $f(x, y) = x^2$ in the PDE. In figure 5(b) we can see that the model captures the dynamics almost exactly, except for the criticality. The variance decreases monotonically as the plate size increases, which is a consequence of the plate spending less time on the boundary. Around the criticality, a deviation exists between experiments and the SVI model, which could be due to the fact that our SVI model is a first-order approximation to the true dynamics, and higher-order effects would appear and dominate the dynamics once the system reaches the criticality of the first-order system.

In the above examples, one has to note that the strength of noise $\sigma = 0.001$ is estimated through the method described in § 2, which only utilizes the data from the trapped state $d/D = 0.8$. Hence figure 5(a,b) is further evidence of the validity of our noise modelling. Moreover, the predictions made above become far worse without the presence of noise, which is a consequence of the stochastic nature of the physics.

The SVI model can also predict quantities that would be hard or infeasible to measure experimentally. When the plate contacts the boundary, a contact force exists due to the fluid force pushing the plate into the wall. From (2.1), the net fluid force on the plate at the right boundary is $F(L, y_t) = \eta w / \lambda \int_{L-d/2}^{L+d/2} v_r dx = -d\eta w / \lambda [\alpha(2d)L - \alpha(d)y_t]$. Without motion, the mean contact force equals the mean fluid force, which can be expressed as $f_c = \lim_{t \rightarrow \infty} \mathbb{E}[F(x_t, y_t) | x_t = L] = -d\eta w / \lambda \{\alpha(2d)L - \alpha(d) \lim_{t \rightarrow \infty} \mathbb{E}[y_t | x_t = L]\}$. Figure 5(c) shows the mean contact force estimated through the simulation – the magnitude of this force is of the order of 0.5 dyne, which would be difficult to measure experimentally. In our setting, we used $f(x, y) = y \mathbf{1}_{\{x=L\}}$ and the formula $\mathbb{E}[y_t | x_t = L] = (\mathbb{E}(f(x_t, y_t))) / (\mathbb{P}(x_t = L))$. The mean contact force has further applications in geophysics, where the contact between continents causes geological events such as mountain formation and earthquakes. Although our SVI model only provides a minimal approximation towards the lab-scale

experiment, the combination of our approach and the analysis of thermal convection with non-trivial boundary conditions (for example, Bakhuis *et al.* 2018) could be adopted to future modelling of geodynamics.

In conclusion, SVIs have been identified as a mathematical framework to model the solid–fluid interactions in thermal convection. In particular, we have noticed that a stochastic version of the ZZ model is well-posed in the framework of a SVI. In contrast to the deterministic model, it is not straightforward to formulate the stochastic problem as a well-posed piecewise dynamics, which is due to the mathematical subtleties inherent to the white noise. SVI theory leads to a solid framework for Kolmogorov equations of the stochastic ZZ model. This allows us to employ the machinery of Markovian tools and thus Kolmogorov equations, and apply them to the modelling of fluid problems. Here we have derived the corresponding Kolmogorov equations and used them for the numerical study of the stochastic ZZ model. From a numerical perspective, techniques based on PDEs are more efficient than Monte Carlo methods for a low-dimensional state variable (dimension ≤ 3).

In coming papers, we plan to investigate several issues such as the property of long-term behaviour and metastable states, applications towards other fluid–structure interaction problems, among others.

Acknowledgements

We thank NSFC grant 11572230, 1161101053, 11601335, 11472106 and NSF grant 1805506, DMS-1463062 for supporting this project.

References

- AHLERS, G., GROSSMANN, S. & LOHSE, D. 2009 Heat transfer and large scale dynamics in turbulent Rayleigh–Bénard convection. *Rev. Mod. Phys.* **81** (2), 503–537.
- ALLSHOUSE, M. R., BARAD, M. F. & PEACOCK, T. 2010 Propulsion generated by diffusion-driven flow. *Nat. Phys.* **6** (7), 516–519.
- BAKHUIS, D., OSTILLA-MNICO, R., VAN DER POEL, E. P., VERZICCO, R. & LOHSE, D. 2018 Mixed insulating and conducting thermal boundary conditions in Rayleigh–Bénard convection. *J. Fluid Mech.* **835**, 491–511.
- BENSOUSSAN, A. & LIONS, J. L. 1982 *Contrôle Impulsionnel et Inéquations Quasi Variationnelles*. Gauthier-Villars.
- BENSOUSSAN, A. & MERTZ, L. 2012 An analytic approach to the ergodic theory of a stochastic variational inequality. *C. R. Mathématique* **350** (7–8), 365–370.
- BENSOUSSAN, A., MERTZ, L., PIRONNEAU, O. & TURI, J. 2009 An ultra weak finite element method as an alternative to a Monte Carlo method for an elasto-plastic problem with noise. *SIAM J. Numer. Anal.* **47** (5), 3374–3396.
- BENSOUSSAN, A., MERTZ, L. & YAM, S. C. P. 2016 Nonlocal boundary value problems of a stochastic variational inequality modeling an elasto-plastic oscillator excited by a filtered noise. *SIAM J. Math. Anal.* **48** (4), 2783–2805.
- BENSOUSSAN, A. & TURI, J. 2008 Degenerate Dirichlet problems related to the invariant measure of elasto-plastic oscillators. *Appl. Math. Opt.* **58** (1), 1–27.
- BENSOUSSAN, A. & TURI, J. 2010 On a class of partial differential equations with nonlocal Dirichlet boundary conditions. In *Applied and Numerical Partial Differential Equations*, pp. 9–23. Springer.
- BENZI, R. 2005 Flow reversal in a simple dynamical model of turbulence. *Phys. Rev. Lett.* **95**, 024502.
- BERNARDIN, F. 2003 Multivalued stochastic differential equations: convergence of a numerical scheme. *Set-Valued Anal.* **11** (4), 393–415.

Dynamics of fluid–structure interaction in turbulent thermal convection

- BROWN, E., NIKOLAENKO, A. & AHLERS, G. 2005 Reorientation of the large-scale circulation in turbulent Rayleigh–Bénard convection. *Phys. Rev. Lett.* **95**, 084503.
- ELDER, J. W. 1968 Convection – the key to dynamical geology. *Sci. Prog.* **56** (221), 1–33.
- FEAU, C., LAURIÈRE, M. & MERTZ, L. 2018 Asymptotic formulae for the risk of failure related to an elasto-plastic problem with noise. *Asymptotic Anal.* **106** (1), 47–60.
- GURNIS, M. 1988 Large-scale mantle convection and the aggregation and dispersal of supercontinents. *Nature* **332** (6166), 695–699.
- GURNIS, M. & ZHONG, S. 1991 Generation of long wavelength heterogeneity in the mantle by the dynamic interaction between plates and convection. *Geophys. Res. Lett.* **18** (4), 581–584.
- HOWARD, L. N., MALKUS, W. V. R. & WHITEHEAD, J. A. 1970 Self-convection of floating heat sources: a model for continental drift. *Geophys. Astrophys. Fluid Dyn.* **1** (1–2), 123–142.
- KRISHNAMURTI, R. & HOWARD, L. N. 1981 Large-scale flow generation in turbulent convection. *Proc. Natl Acad. Sci. USA* **78** (4), 1981–1985.
- LAURIÈRE, M. & MERTZ, L. 2015 Penalization of a stochastic variational inequality modeling an elasto-plastic problem with noise. *ESAIM: Proc. Surv.* **48**, 226–247.
- LIONS, P.-L. & SZNITMAN, A.-S. 1984 Stochastic differential equations with reflecting boundary conditions. *Commun. Pure Appl. Maths* **37** (4), 511–537.
- LOWMAN, J. P. & JARVIS, G. T. 1993 Mantle convection flow reversals due to continental collisions. *Geophys. Res. Lett.* **20** (19), 2087–2090.
- LOWMAN, J. P. & JARVIS, G. T. 1995 Mantle convection models of continental collision and breakup incorporating finite thickness plates. *Phys. Earth Planet. Inter.* **88** (1), 53–68.
- MERCIER, M. J., ARDEKANI, A. M., ALLSHOUSE, M. R., DOYLE, B. & PEACOCK, T. 2014 Self-propulsion of immersed objects via natural convection. *Phys. Rev. Lett.* **112** (20), 204501.
- MERTZ, L. & BENSOUSSAN, A. 2015 Degenerate Dirichlet problems related to the ergodic property of an elasto-plastic oscillator excited by a filtered white noise. *IMA J. Appl. Maths* **80** (5), 1387–1408.
- MERTZ, L. & FEAU, C. 2012 An empirical study on plastic deformations of an elasto-plastic problem with noise. *Prob. Engng Mech.* **30**, 60–69.
- MERTZ, L., STADLER, G. & WYLIE, J. 2017 A backward Kolmogorov equation approach to compute means, moments and correlations of non-smooth stochastic dynamical systems. Preprint, [arXiv:1704.02170](https://arxiv.org/abs/1704.02170).
- SREENIVASAN, K. R., BERSHADSKII, A. & NIEMELA, J. J. 2002 Mean wind and its reversal in thermal convection. *Phys. Rev. E* **65**, 056306.
- TURCOTTE, D. L. & SCHUBERT, G. 2002 *Geodynamics*. Cambridge University Press.
- WHITEHEAD, J. A. 1972 Moving heaters as a model of continental drift. *Phys. Earth Planet. Inter.* **5**, 199–212.
- WHITEHEAD, J. A. & BEHN, M. D. 2015 The continental drift convection cell. *Geophys. Res. Lett.* **42** (11), 4301–4308.
- ZHANG, J. & LIBCHABER, A. 2000 Periodic boundary motion in thermal turbulence. *Phys. Rev. Lett.* **84** (19), 4361.
- ZHONG, J.-Q., STERL, S. & LI, H.-M. 2015 Dynamics of the large-scale circulation in turbulent Rayleigh–Bénard convection with modulated rotation. *J. Fluid Mech.* **778**, R4.
- ZHONG, J.-Q. & ZHANG, J. 2005 Thermal convection with a freely moving top boundary. *Phys. Fluids* **17** (11), 115105.
- ZHONG, J.-Q. & ZHANG, J. 2007a Dynamical states of a mobile heat blanket on a thermally convecting fluid. *Phys. Rev. E* **75** (5), 055301.
- ZHONG, J.-Q. & ZHANG, J. 2007b Modeling the dynamics of a free boundary on turbulent thermal convection. *Phys. Rev. E* **76** (1), 016307.